

**ENGR 350 - Mechanics of Materials, Fall 2013**

**Homework #4**

**Due: Mon., Sept. 22**

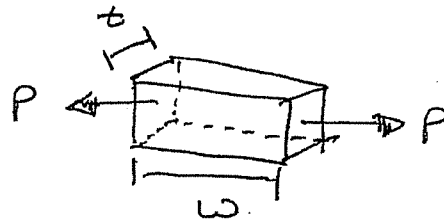
1. Problem P4.1, p. 79
2. Problem P4.4, p. 79
3. Problem P4.6, p. 80
4. Problem P4.11, p. 81

*(all problems are from the textbook).*

4.1

$$w = 25 \text{ mm} \quad t = 16 \text{ mm}$$

$$P = 145 \text{ kN}$$



$$a) \quad \sigma_y = 550 \text{ MPa}$$

$$\sigma = \frac{F}{A} = \frac{(145 \text{ kN})(1000 \text{ N/kN})}{(25 \text{ mm})(16 \text{ mm})}$$
$$= 362.5 \text{ MPa}$$

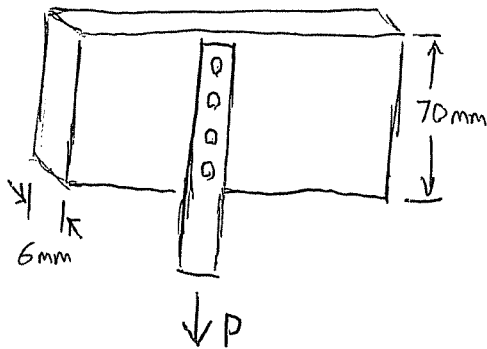
$$FS = \frac{\sigma_y}{\sigma_{act}} = \frac{550 \text{ MPa}}{362.5 \text{ MPa}} = \boxed{1.517}$$

$$B) \quad \sigma_u = 1100 \text{ MPa}$$

$$FS = \frac{\sigma_u}{\sigma_{act}}$$

$$= \frac{1100 \text{ MPa}}{362.5 \text{ MPa}} = \boxed{3.03}$$

4.4) < Given >



Screws:

$$\sigma_s = 165 \text{ MPa} = \tau_{us}$$

$$SF_u = 3.0$$

Metal Bar:

$$\sigma_y = 250 \text{ MPa}$$

$$\sigma_u = 350 \text{ MPa}$$

$$SF_B = 3.0$$

$$SF_y = 1.67$$

< Goal >

Allowable load P

< Solution >

Bar (normal stress):

$$\sigma_{\text{allow}} = \frac{\sigma_y}{SF_y} = \frac{250 \text{ MPa}}{1.67} = 149.7 \text{ MPa}$$

$$\begin{aligned} P_{\text{allow}} &= \sigma_{\text{allow}} A_{\text{bar}} \\ &= 149.7 \text{ MPa} (70 \text{ mm})(6 \text{ mm}) \\ &= 62,874 \text{ N} \end{aligned}$$

Lag screws (shear stress):

$$\tau_{\text{allow}} = \frac{\tau_{us}}{SF_u} = \frac{165 \text{ MPa}}{3.0} = 55.0 \text{ MPa}$$

$$A_{\text{screw}} = \pi d^2 / 4 = \pi (22 \text{ mm})^2 / 4 = 380.1 \text{ mm}^2$$

$$V_{\text{screw}} = (55 \text{ N/mm}^2)(380.1 \text{ mm}^2) = 20,907 \text{ N}$$

$$P_{\text{allow}} = 4 \text{ screws} (20,907 \text{ N/screw}) = 83,630 \text{ N}$$

Bar (bearing stress):

$$\sigma_{\text{allow}} = \frac{\sigma_y}{SF_B} = \frac{\sigma_y}{SF_B} = \frac{350 \text{ MPa}}{3.0} = 116.67 \text{ MPa}$$

$$F_B = 116.7 \text{ N/mm}^2 (22 \text{ mm})(6 \text{ mm}) = 15,400 \text{ N}$$

$$P_{\text{allow}} = 4F_B = 4(15,400 \text{ N}) = 61,600 \text{ N}$$

Allowed Load:

$$P_{\text{allow}} = 61,600 \text{ N} = \boxed{61.6 \text{ kN}}$$

3-0285 — 50 SHEETS — 5 SQUARES  
 3-0286 — 100 SHEETS — 5 SQUARES  
 3-0287 — 200 SHEETS — 5 SQUARES  
 3-0187 — 200 SHEETS — FILLER

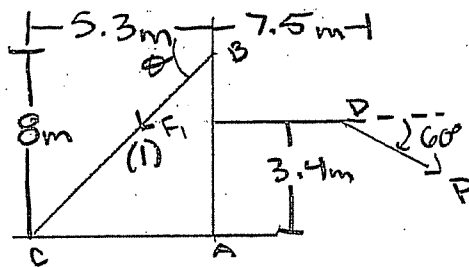
COMET

### Problem 4.6

Given: Rigid structure supported by  $D_1 = 35\text{mm}$  rod  
 tie rod connected at B and C  $D_2 = 30\text{mm}$  pin  
 by  $D = 30\text{mm}$  double shear  $\text{single shear}$   
 $\sigma_{t1} = 250\text{MPa}$   $\sigma_u = 330\text{MPa}$   $P = 50\text{kN}$

- Find:
- normal stress in rod 1
  - shearing stress in pins at A and B
  - FOS  $\rightarrow$  tie rod 1
  - FOS pins A and B

Solution:



a)

$$\tan \theta = \frac{8\text{m}}{5.3\text{m}}$$

$$= 1.50943$$

$$\theta = 56.4755^\circ$$

$$\sum F_x = P \cos 60^\circ - F_1 \cos 56.4755^\circ + A_x$$

$$\sum F_y = -P \sin 60^\circ - F_1 \sin 56.4755^\circ + A_y = 0$$

$$\sum M_A = (F_1 \cos 56.475^\circ)(8\text{m}) - (P \cos 60^\circ)(3.4\text{m}) - P \sin 60^\circ(7.5) = 0$$

$$F_1 = \frac{P(3.4\text{m}) \cos 60^\circ + (7.5) \sin 60^\circ}{8\text{m}(\cos 56.4755^\circ)}$$

$$= \frac{50(3.4) \cos 60^\circ + (7.5)(\sin 60^\circ)}{8(\cos 56.475^\circ)}$$

$$= 92.7405\text{ kN}$$

$$A_x = F_1 \cos 56.4755 - P \cos 60$$

$$= (92.7405) \cos 56.4755 - 50 (\cos 60)$$

$$= 26.2199 \text{ kN}$$

$$A_y = F_1 \sin 56.4755 + P \sin 60$$

$$= (92.7405) \sin(56.4755) + 50 \sin 60$$

$$= 120.6144 \text{ kN}$$

$$|A| = \sqrt{A_x^2 + A_y^2}$$

$$= \sqrt{(26.2199)^2 + (120.6144)^2}$$

$$= 123.4314 \text{ kN}$$

$$A_1 = \frac{\pi}{4} (35)^2$$

$$= 962.1127 \text{ mm}^2$$

$$\sigma_1 = \frac{F_1}{A_1}$$

$$= \frac{92740.5 \text{ N}}{962.1127 \text{ mm}^2}$$

$$= 96.39 \text{ MPa}$$

$$\text{b) } A_{v, A} = \frac{\pi}{4} (30)^2$$

$$= 706.858 \text{ mm}^2$$

$$\tau_A = \frac{123431.4}{706.8583}$$

$$= 174.619 \text{ MPa}$$

3-0235 — 50 SHEETS — 5 SQUARES  
3-0236 — 100 SHEETS — 5 SQUARES  
3-0237 — 200 SHEETS — 5 SQUARES  
3-0137 — 200 SHEETS — FILLER

COMET

$$Z_B = \frac{92740.5}{904.7786}$$

$$= 102.5008 \text{ MPa}$$

$$\text{c) } FS_1 = \frac{Z_U}{Z_A}$$
$$= \frac{250 \text{ MPa}}{96.39 \text{ MPa}}$$

$$= 2.59$$

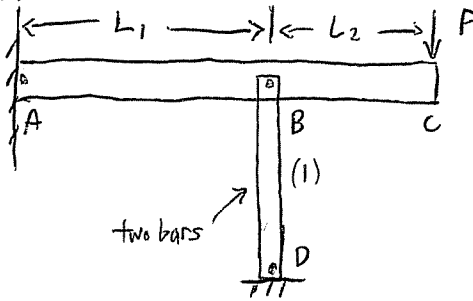
$$\text{d) } FS_A = \frac{Z_U}{Z_A}$$
$$= \frac{330 \text{ MPa}}{174.6197 \text{ MPa}}$$

$$= 1.89$$

$$FS_B = \frac{Z_U}{Z_B}$$
$$= \frac{330 \text{ MPa}}{102.5008 \text{ MPa}}$$

$$= 3.22$$

4.11) (Given)



$$L_1 = 36 \text{ in}$$

$$L_2 = 24 \text{ in}$$

$$d_A = d_B = d_D = 0.5 \text{ in}$$

$$\sigma_y = 36 \text{ Ksi (bars)}$$

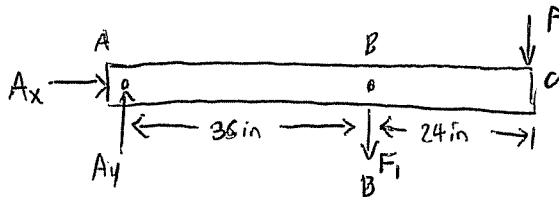
$$\sigma_u = 72 \text{ Ksi (pins)} = T_u$$

$$SF = 3.0$$

< Goal >

Allowable load P

< Solution >



$$\pm \sum F_x = A_x = 0$$

$$+\uparrow \sum F_y = A_y - P - F_1 = 0$$

$$\curvearrowright \sum M_A = -L_1 F_1 - (L_1 + L_2) P = 0$$

$$F_1 = -\left(\frac{L_1 + L_2}{L_1}\right) P = -\left(\frac{36 \text{ in} + 24 \text{ in}}{36 \text{ in}}\right) P = -1.667 P$$

$$A_y = P + F_1 = P + (-1.667 P) = 0.667 P$$

$$A = \sqrt{A_x^2 + A_y^2} = \sqrt{(0)^2 + (0.667 P)^2} = 0.667 P$$

strut (1):

$$\sigma_{\text{allow}} = \frac{\sigma_y}{SF} = \frac{36 \text{ Ksi}}{3.0} = 12.0 \text{ Ksi}$$

$$F_{\text{allow}} = \sigma_{\text{allow}} A = 12 \text{ Ksi} (2 \times 2 \text{ in} \cdot 0.25 \text{ in}) = 12.0 \text{ Kips}$$

$$P_{\text{max}} \leq \frac{F_{\text{allow}}}{1.667} = \frac{12.0 \text{ kips}}{1.667} = 7.20 \text{ Kips}$$



Pins:

$$T_{\text{allow}} = \frac{T_u}{SF} = \frac{72 \text{ ksi}}{3.0} = 24.0 \text{ ksi}$$

$$A_{\text{pin}} = \pi (0.5 \text{ in})^2 / 4 = 0.1964 \text{ in}^2$$

Pin A:

$$A_{\text{allow}} = T_{\text{allow}} A_{\text{pin}} = 24.0 \text{ ksi} (0.1964 \text{ in}^2) = 4.7124 \text{ kips}$$

$$P_{\text{max}} \leq \frac{A_{\text{allow}}}{0.667} = \frac{4.7124 \text{ kips}}{0.667} = 7.07 \text{ kips}$$

Pin B and Pin D:

$$F_{\text{allow}} = T_{\text{allow}} A_{\text{pin}} = 24 \text{ ksi} (2) (0.1964 \text{ in}^2) = 9.425 \text{ kips}$$

$$P_{\text{max}} \leq \frac{F_{\text{allow}}}{1.667} = \frac{9.425 \text{ kips}}{1.667} = 5.655 \text{ kips} \Rightarrow \underline{\text{lowest value}}$$

Maximum load:

$$P_{\text{max}} = 5.6549 \text{ kips} = \boxed{5.65 \text{ kips}}$$